Modeling Sprint Cycling Using Field-Derived Parameters and Forward Integration

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ABSTRACT

MARTIN, J. C., A. S. GARDNER, M. BARRAS, and D. T. MARTIN. Modeling Sprint Cycling Using Field-Derived Parameters and Forward Integration. Med. Sci. Sports Exerc., Vol. 38, No. 3, pp. 592–597, 2006. We previously reported that a mathematical model could accurately predict steady-state road-cycling power when all the model parameters were known. Application of that model to competitive cycling has been limited by the need to obtain accurate parameter values, the non-steady-state nature of many cycling events, and because the validity of the model at maximal power has not been established. Purpose: We determined whether modeling parameters could be accurately determined during field trials and whether the model could accurately predict cycling speed during maximal acceleration using forward integration. Methods: First, we quantified aerodynamic drag area of six cyclists using both wind tunnel and field trials allowing for these two techniques to be compared. Next, we determined the aerodynamic drag area of three world-class sprint cyclists using the field-test protocol. Track cyclists also performed maximal standing-start time trials, during which we recorded power and speed. Finally, we used forward integration to predict cycling speed from power-time data recorded during the maximal trials allowing us to compare predicted speed with measured speed. Results: Field-based values of aerodynamic drag area (0.258 ± 0.006 m²) did not differ (P = 0.53) from those measured in a wind tunnel (0.261 ± 0.006 m²). Forward integration modeling accurately predicted cycling speed (y = x, r² = 0.989) over the duration of the standing-start sprints. Conclusions: Field-derived values for aerodynamic drag area can be equivalent to values derived from wind tunnel testing, and these values can be used to accurately predict speed even during maximal-power acceleration by world-class sprint cyclists. This model could be useful for assessing aerodynamic issues and for predicting how subtle changes in riding position, mass, or power output will influence cycling speed. Key Words: WORLD CLASS, AERODYNAMICS, WIND TUNNEL, MAXIMAL POWER

Mathematical cycling models (4,5,8,12,14) allow scientists, coaches, and athletes to systematically examine the extent to which alterations to various aspects of the cyclist, his or her bicycle, or environmental conditions might alter cycling performance. Although individual models include specific nuances, most include terms for power produced by the cyclist and power required to overcome aerodynamic drag, rolling resistance, drive train friction, and to accelerate or raise the cyclist’s center of mass. The relative importance of each of those terms depends on the instantaneous conditions, but for steady-state riding over relatively flat terrain, aerodynamic drag has been reported to be the dominant term, requiring up to 96% of the cyclist’s power (12). Consequently, minimizing aerodynamic drag is paramount in efforts to optimize cycling performance. For other conditions, such as maximal acceleration from a standing start or cycling up steep grades, changes in kinetic or potential energy will consume most of the cyclist’s power (8). Furthermore, a cyclist can experience several conditions within a single competitive event. During a 1-km track cycling time trial, for example, power produced at the start will mostly act to accelerate the rider, whereas power produced later in the race will mostly act to overcome aerodynamic drag. Thus, decisions regarding the relative importance of aerodynamic drag and body or bicycle mass, and the timing of application of power (i.e., pacing strategies) require careful analysis that can be facilitated with mathematical modeling.

Previously, we (12) reported that a mathematical model could account for 97% of the variability in steady-state road cycling power when all the model parameters (aerodynamic drag, rolling resistance, friction in the bearings and chain drive system, and changes in kinetic and potential energy) were known. Measuring these parameters, however, requires sophisticated wind tunnel (12), tire (9), and bearing testing (3). Additionally, although our model predicted cycling power quite well, our measurement intervals were fairly large (472 m) and our model did not account for rapid changes in power or speed. Finally, as others have done (13,14), we focused on modeling endurance cycling (~200 W) rather than sprint cycling where maximal power can exceed 1000 W. It is worth pointing out that of the eight
performed three discrete tasks. First, we modified our maximal-power cycling. More specifically, we designed a based parameters and by applying it to non-steady-state, many cycling events, and because the validity of the model thus, whereas our previous model has been useful (8,10,t 1), the application has been limited by the need to obtain accurate parameter values, the non-steady-state nature of many cycling events, and because the validity of the model at maximal power has not been established.

In this investigation, our goal was to broaden the application of our previous model by adapting it to field-based parameters and by applying it to non-steady-state, maximal-power cycling. More specifically, we designed a series of experiments to determine whether model parameters could be accurately quantified during field tests, and whether a model, using field-based parameters, could accurately predict cycling speed during high-power, non-steady-state cycling. We hypothesized that values for aerodynamic drag could be accurately derived from field test data and that speed during non-steady-state cycling could be accurately modeled using a simple linear forward integration technique.

**METHODS**

Overview. To accomplish our stated objectives we performed three discrete tasks. First, we modified our previous model so that it depended only on two coefficients: aerodynamic drag area (coefficient of drag × frontal area; C_dA) and a global coefficient of friction (µ). Second, we determined those coefficients from field test data and compared them with C_dA values measured in a wind tunnel and with previously reported values for coefficients of rolling resistance (C_RR). Third, we used field-derived coefficients and forward integration to predict cycling speed from known power–time data and compared predicted speed with measured speed to establish the validity of the forward integration technique. For these final trials, we recorded data during maximal standing-start accelerations performed by world-class sprint cyclists.

Task 1: Model modification. In our previous model (12), we expressed power as a function of aerodynamic drag, rolling resistance, bearing friction, changes in kinetic and potential energy, and drive system efficiency. Each term in the model was a function of either ground speed (S_g) or the product of air speed squared and ground speed (S_g^2 S_p^2). Consequently, we rewrote our original equation to include only two coefficients:

\[
\text{Power} = \left[ C_d A \times \left( \frac{1}{2} \rho S_a^2 S_g^2 \right) + \frac{1}{2} S_g F_W \right] \left( S_g \right)^2 + \frac{1}{2} S_g F_W \]  

in which C_dA represents the combined effective frontal area of the bicycle and rider, and of the wheel spokes (C_dA and F_W in our previous model), µ represents a global coefficient of friction (including C_RR, we assumed to be 97.7% based on our previous findings (12)), PE is potential energy, KE is kinetic energy, p is air density, and F_N is the normal force exerted by the bicycle tires on the rolling surface (essentially weight of the bicycle and rider). Equation 1 was rearranged to form an expression with only the two resistance terms on one side and the power and energy terms on the other:

\[
P \times E = -\Delta \text{PE}/\Delta t - \Delta \text{KE}/\Delta t = C_d A \times \left( \frac{1}{2} \rho S_a^2 S_g^2 \right) + \frac{1}{2} \left( S_g F_W \right) \]  

Using this form of the equation, we can determine C_dA and µ via regression analysis.

Task 2: Parameter determination and comparison. In our previous study (12), we obtained C_dA of six male cyclists from wind tunnel testing conducted at an air speed of approximately 13.4 m·s⁻¹ and at yaw angles (i.e., the angle of alignment between the bicycle and the air stream) of 0, 5, 10, and 15°. In that previous study, each cyclist also performed cycling trials on a straight concrete surface with a grade of 0.3%, during which we measured bicycle ground speed, cycling power, air speed, and wind direction. Each cyclist rode the test section in both directions at three different speeds (approximately 7, 9, and 11 m·s⁻¹), and one subject rode at a fourth speed (12 m·s⁻¹).

In the present study, we reused those previously obtained data to produce field-derived values for C_dA and µ. Specifically, we used those power, ground speed, wind speed, air density, and energy data, and the assumed value for drive system efficiency, to determine field-based C_dA and µ values in a two-step process. First, we determined a value for the quantities on the left side of Equation 2 (P × E = ΔPE/Δt = ΔKE/Δt) and for several terms on the right side of the equation (1/2 ρ S_a² S_g² and S_g F_W) for each trial, and determined C_dA and µ for each subject via multiple linear regression. We then entered the mean value for µ into Equation 2 and determined individual C_dA values via linear regression. We used this two-step process to establish a single value for µ, which we expected to be constant for a specific bicycle tire and surface combination.

Task 3: Forward integration modeling of non-steady-state cycling performance. Three world-class sprint cyclists (subject 1, a male match-sprint specialist: 1.83 m, 96 kg; subject 2, a male kilometer time-trial specialist: 1.82 m, 87 kg; and subject 3, a female 500-m specialist: 1.65 m, 68 kg) volunteered to participate in this phase of the investigation. Each subject had been world champion at least once in a track sprint cycling event, and all were Olympic medalists. We explained the requirements of the investigation to each cyclist, and she or he gave written informed consent. The Australian Institute of Sport human research ethics committee approved the methods used in this investigation. These three cyclists only performed field trials; they did not participate in wind tunnel testing. Each cyclist performed steady-state trials (6–21 s) on an indoor velodrome at speeds ranging from 6 to 16 m·s⁻¹ in seated and standing positions while power and speed were recorded.
at a frequency of 5 Hz with a dynamically calibrated professional version Schoberer Resistance Mechanism (SRM) Powermeter (Germany). We determined the modeling coefficients $C_{DA}$ and $\mu$ from the steady-state data, as described above. The steady-state trials used to calculate $C_{DA}$ and $\mu$ were performed on a 250-m velodrome (Superdrome, Adelaide, South Australia) that is approximately oval in shape. When cycling through a turn, the rider’s center of mass travels a shorter path than the wheels because of the lean angle and, thus, moves at a reduced speed. To account for that difference during the calculation of $C_{DA}$ and $\mu$, we assumed that the track was circular with a circumference of 250 m, and that the wheels traveled a circular path with a radius that was greater than that of the center of mass by the height of the center of mass multiplied by the sine of the lean angle (where lean angle = arctangent [centripetal acceleration/gravity]). We assumed that the height of the center of mass was equal to the height of the top of the saddle. Those assumptions would not provide accurate results at any specific point, but should provide a realistic approximation for the average data. We determined the normal force on the track surface as the vector sum of weight and centripetal force (using the circular-track assumption) and used the value associated with each speed in the regression procedure.

Each subject performed a maximal standing-start time trial of a length specific to her or his competitive event. During each trial, power and speed were recorded using the same SRM Powermeter as in the steady-state trials. Power and speed data, together with field-derived $C_{DA}$ and $\mu$, were used to evaluate the accuracy of linear forward integration modeling in which the conditions at one point in time are used to predict conditions at a subsequent point in time. Initial conditions were established using the first registered power values that were recorded once the pedaling rate exceeded 30 rpm (7). Using Equation 2, we modeled the power required to maintain the initial speed and assumed that the difference between required power and measured power (excess power) produced acceleration ($a = \frac{\text{excess power}}{\text{speed} \times \text{mass}}$). The increase in speed that should occur in the time from one data point to the next is the product of acceleration and change in time. Initial conditions were established using the first registered speed, acceleration, and data collection frequency: $S_{n+1} = S_n + a \times t$. From point 2 forward, only predicted speed and measured power values were used in the model. To account for the moment of inertia of the two wheels we added the mass of two tires and two rims (~1 kg) to total mass of body and bicycle. As described above, the wheels move faster than the center of mass when cycling around a turn. We predicted the speed of the wheels based on lean angle, which was determined from the speed of the center of mass and track geometry. We assumed that the center of mass of the bike–rider system was located at the height of the saddle. If that assumption were incorrect, the model would incorrectly predict speed in the turns. Track geometry was modeled as a straight away and two constant radius turns ($r = 20.7$ m for the 200-m track used by one cyclist, and $r = 27$ m for the 250-m track used by the other two cyclists), and the radius was determined from the known length of the track and of the straight. This technique produced discontinuities in predicted speed of the wheels at the entrance and exit of each turn. We used a moving average (1 s) of the sine of the lean angle to account for the transitions and provide continuous wheel speed predictions. Finally, we used our model to evaluate the performance improvements that might be realized by decreasing mass by 2%, decreasing $C_{DA}$ by 2%, decreasing both mass and drag by 2%, and by decreasing mass and power by 2%.

**Statistics.** $C_{DA}$ values determined from wind tunnel testing were compared with those determined from field trials with a paired Student’s $t$-test. The relationships of modeled and measured speed for the maximal standing-start time trials were determined with linear regression. The 95% confidence interval for Pearson’s correlation coefficient was determined using the method of Fisher (6). A Bland–Altman plot was generated to allow inspection of the modeled and measured speed data.

**RESULTS**

**Parameter determination and comparison.** The $C_{DA}$ values determined from wind tunnel testing for each subject, interpolated to represent the yaw angles encountered during the cycling trials, averaged $0.261 \pm 0.006$ m$^2$, and individual values are shown in Table 1. Field-derived values of $C_{DA}$ averaged $0.258 \pm 0.006$ m$^2$ and did not differ ($P = 0.53$) from wind tunnel values (Table 1). The field-derived global coefficient of friction ($\mu$) while cycling on the taxiway was $0.0043 \pm 0.0006$.

**Forward integration modeling of non-steady-state cycling performance.** Field-derived values of $C_{DA}$ for the sprint cyclists were $0.245 \pm 0.044$ m$^2$ for the seated position and $0.304 \pm 0.055$ m$^2$ for the standing position, and individual values are reported in Table 2 and shown in Figure 1 as a sample data set of power versus speed. Subjects 2 and 3 used aerodynamic handlebars with elbow support for the seated position, whereas subject 1 used traditional racing handlebars. The mean value for $\mu$ was $0.0025 \pm 0.001$. Power, measured speed, and modeled speed for each subject during his or her time trial, which was a length appropriate to his or her specialization, are shown in Figure 2 (250 m for subject 1, a male match-sprint.
TABLE 2. Drag area.

<table>
<thead>
<tr>
<th></th>
<th>Seated</th>
<th>Standing</th>
</tr>
</thead>
<tbody>
<tr>
<td>Subject 1</td>
<td>0.352</td>
<td>0.414</td>
</tr>
<tr>
<td>Subject 2</td>
<td>0.215</td>
<td>0.245</td>
</tr>
<tr>
<td>Subject 3</td>
<td>0.186</td>
<td>0.252</td>
</tr>
</tbody>
</table>

Drag area values of the three world-class cyclists who participated in the second phase of this study. Note that subject 1 used conventional handlebars, whereas subjects 2 and 3 used aerodynamic handlebars with elbow supports.

Our finding that \( C_{DA} \) values determined in field trials were nearly identical to those determined in the wind tunnel is important because access to wind tunnel testing is both limited and expensive. Our data demonstrate that 5–10 steady-state cycling trials can be used to reproduce \( C_{DA} \) values obtained from wind tunnel testing. The mathematics we described are not particularly sophisticated. Our modeling technique can be used to quantify and possibly minimize drag area. Additionally, our model could be used to predict performance changes that might result from changes in various parameters (e.g., mass, equipment, frontal surface area).

FIGURE 1—Example of a power–speed data set. Power and speed were used to determine \( C_{DA} \) and \( \mu \) via multiple linear regression.

FIGURE 2—Power, measured speed, and predicted speed. Power (open circles) and speed (black line) were measured during maximal standing-start time trials performed by three world-class cyclists, and speed was predicted (gray line) using forward integration. Panel A displays the data from a 250-m time trial (subject 1), panel B displays the data from a 1000-m time trial (subject 2), and panel C displays the data from a 500-m time trial (subject 3). Our model accurately predicted speed throughout the trials (\( y = x, r^2 = 0.989, \text{SEE} = 0.25 \text{ m/s}^{-1} \)).
Thus, the drag areas of subjects 1 and 2 differed by 68% when standing (i.e., effect of aerodynamic handlebars would be absent) even though their body mass differed by only 10%, and predicted body surface area (15) differed by only 4% (2.18 m$^2$ vs 2.09 m$^2$). Consequently, the dramatic difference in drag area between these two world-class cyclists is not simply caused by body mass or surface area. Future research involving drag area and full anthropometric descriptions may provide a means for predicting body types with low drag area.

Our linear forward integration technique produced remarkably accurate predictions of speed during maximal standing-start acceleration and fatigue-related deceleration. The model produced similar predictive accuracy with cyclists who produced peak power outputs from 1377 to 2517 W and performed time trials from 250 to 1000 m in length. The accuracy was particularly satisfying because of the simple nature of the model and because of the capacities of our subjects. Specifically, our subjects were among the most successful track sprint cyclists in the world, and thus their acceleration likely represents the upper limits of human performance and a most severe modeling challenge. Even so, our modeling technique accounted for 99% of the variation in actual speed and predicted speed with a standard error of 0.25 m/s$^{-1}$. Such high precision in these extreme conditions suggests that our model will accurately predict cycling speed in most non-steady-state situations.

Modeling speed during velodrome cycling presented several unique challenges. First, the speed of the center of mass differed from the speed of the wheels when cycling in the turns because of lean angle. We modeled the velodrome with constant radius curves and straight lines. Such simple geometry almost certainly does not represent actual velodrome geometry; however, our attempts to obtain actual specifications from the track designers failed. Even so, our simple geometry was adequate to model speed with minimal errors. In addition, the center of aerodynamic pressure may not have been at the same position as the center of mass, and thus the appropriate air speed may have differed from the speed of the center of mass while in the turns. We did not attempt to account for this potential difference in our model. Indeed, we are unaware of any published values for the height of the center of pressure. Centripetal force increased normal force while cycling in the turns. To account for that increase, we modeled normal force (which determines rolling resistance) as the vector sum of gravity and centripetal force.

One particularly intriguing aspect of these data was the range of drag area values of the two male track sprint cyclists. Subject 1 was a match-sprint specialist with a body mass of 96 kg who used traditional racing handlebars. His drag area values (0.332 m$^2$ seated and 0.414 m$^2$ standing) were much greater than those for subject 2 (87 kg) who used aerodynamic handlebars and exhibited drag area values of 0.215 m$^2$ when seated and 0.245 m$^2$ when standing.

### TABLE 3. Modeled scenarios.

<table>
<thead>
<tr>
<th>Predicted Time Changes (s)</th>
<th>Reduced Mass</th>
<th>Reduced Drag</th>
<th>Reduced Mass and Drag</th>
<th>Reduced Mass and Power</th>
</tr>
</thead>
<tbody>
<tr>
<td>Subject 1 250 m</td>
<td>-0.091</td>
<td>-0.000</td>
<td>-0.091</td>
<td>+0.031</td>
</tr>
<tr>
<td>Subject 2 1000 m</td>
<td>-0.120</td>
<td>-0.314</td>
<td>-0.456</td>
<td>+0.320</td>
</tr>
<tr>
<td>Subject 3 500 m</td>
<td>-0.093</td>
<td>-0.120</td>
<td>-0.214</td>
<td>+0.123</td>
</tr>
</tbody>
</table>

Predicted time changes (s) for each trial for four scenarios: 2% decrease in mass, 2% decrease in aerodynamic drag, 2% reduction in both drag and mass, and 2% reductions in both mass and power. For each subject, the time changes are specific to her or his competition distance. A negative sign indicates reduction in performance time (improved performance), and a positive sign indicates increased performance time (decreased performance).
acting along the longitudinal axis of the rider. Velodrome surfaces are banked, even on the straight portions of the track. Because of this banking, the tires were subjected to side loading, which potentially increased the rolling resistance. We did not model the side loading in any specific way, but it is likely to contribute to our value for $\mu$ (as previously discussed). Finally, our velodrome model did not provide gradual transitions between the straights and the turns. We modeled the transitions as a 1-s (5 point) moving average in the sine of the lean angle. Although this transition did not exactly duplicate the speed changes at each turn entry and exit, it served as a reasonable approximation as shown in Figure 2.

One of the most important applications of our model may be in predicting the effects of changes in model parameters on cycling performance. With this in mind, we modeled four scenarios for each subject: 2% reduction in mass, 2% reduction in $C_pA$, 2% reductions in drag and mass, and 2% reductions in mass and power (Table 3). The reduced mass was predicted to give a greater performance advantage for subject 1 in the 250-m time trial, whereas the reduced drag produced a larger performance increase for subjects 2 and 3 in the 500- and 1000-m time trials, respectively. Additionally, the predicted performance changes related to mass and drag were approximately additive, suggesting that mass and aerodynamic drag influence performance via different mechanisms such as initial acceleration (related to mass) and maximal speed (related to aerodynamic drag). Finally, the combination of reduced mass and power resulted in increased performance time for all three athletes, which underscores the potential dangers associated with reducing weight at the risk of decreasing power. Although these changes in performance time may seem small, they are large enough to have a substantial impact on final standings in world-class competition. For example, at the 2005 Track Cycling World Championships, the first three finishers in the women’s 500-m time trial were separated by 0.19 s, which is less than the difference our model predicts for a 2% decrease in aerodynamic drag. Similarly, the top two finishers in the men’s 1000-m time trial at that event were separated by only 0.065 s, which is less of an advantage than our model predicts for either the reduced mass or drag. Thus, the small but important improvements in performance predicted by our model should be given serious consideration by athletes, coaches, and sport scientists.

In summary, we have demonstrated that field-derived values for modeling coefficients were equivalent to those derived from sophisticated testing and that those coefficients can be used to accurately predict speed during maximal-power, non-steady-state cycling using linear forward integration. We believe that this modeling procedure will allow cyclists, coaches, and sport scientists to perform aerodynamic testing, optimization, and modeling without the use of a wind tunnel. Additionally, by using forward integration of known power–time profiles, cyclists can make realistic predictions of potential performance benefits or decrements from variations in any of the model parameters.

This study was only possible because of the cooperation of many people across three continents. We would like to thank the world champion athletes who donated their valuable training time, Prof. Allan Hahn and the staff in the physiology department at the AIS, Steve Hed who provided funding for the wind tunnel testing, and John Cobb for his technical assistance in the wind tunnel.

REFERENCES


Aerodynamic Drag Area of Cyclists Determined with Field-Based Measures

James C Martin, A Scott Gardner, Martin Barras, David T Martin

Aerodynamic drag is an important factor in the performance of competitive track and road cyclists. Recently we used wind-tunnel testing to validate a practical measure of aerodynamic drag derived from a field test. We present here instructions for performing the field test on a straight flat road or in a velodrome, and we include a spreadsheet for performing the calculations.

KEYWORDS: model, performance, test, track, velodrome

These instructions pertain to a spreadsheet that will allow you to determine aerodynamic drag area (CdA) from six field trials. Data must be recorded by an SRM or other accurate power measuring device. Please refer to our recent paper for additional details (Martin et al., 2006). Note that in the paper we used a two-step process to first determine friction (μ) and then to determine CoA. This process worked quite well with a large number of samples (several data points from many riders) but μ tends to be unstable when evaluated with a few points. In the current spreadsheet we have given some typical values for μ that will facilitate more robust calculation of CdA. While this approach will introduce some error, it will likely be a small error in a small term, so the overall impact on CdA should be negligible.

General Instructions

Start by going to the Air Density Calculator tab. Enter a value for temperature in degrees Celsius in cell A2. Enter a value for relative humidity in percent in cell B2. Enter a value for barometric pressure in cell C2. Barometric pressure must be in units of millibars and the values must be absolute. This may be difficult to determine because weather stations often report barometric pressure values corrected to sea level. The most direct way to determine barometric pressure would be for you to have a barometer on site during your testing. These can be purchased for as little as US$30. Alternatively, if you are in the USA you can go to the website and find a station near your testing site. You will need to set the output to metric units.

Both the road and track calculations must be based on six trials. Do not leave cells blank. If you cannot do six trials, you could enter data for some of the trials twice, but that is not ideal. Each trial should be done a constant speed throughout the test section. The range of speeds you use should be as wide as possible, from say 10 km/h to as high a speed as the rider can hold steady for the test section. By setting the speedometer to km/h you will get maximum resolution. You will need to convert the speed to meters per second for the spreadsheet calculations.

Instructions for a Straight Flat Road

We have not specified a length for the "test section" but we recommend as great a length as you can find that is straight and flat. We have used 470 m with very good results.

Start by going to the CdA for a straight road tab. You will need to enter the following values for each trial:
1. Column A: Bike and rider mass in kg.
2. Column B: The road grade in percent. Note that this will depend on the direction. Positive grade denotes uphill and negative grade denotes downhill. These values can be obtained from local government agencies. Ask for "as built plans" or "as built surveys".
3. Column C: Wind velocity in meters per
second must be entered for each trial. To enter the proper value you will need to know the wind velocity and direction as well as the direction of the road. From those data, you must determine the component of the wind that is parallel with the road. We will assume that you can make those calculations and will not attempt to explain them. Negative values denote wind in the same direction as the bicycle is traveling and positive values denote wind that opposes the rider.

4. Column D and E: Velocity and the beginning (initial) and end (final) of the measurement interval in meters per second. These can be read off of the SRM data but you must know the time associated with the start and end.

5. Column F: Average velocity over the measured section in meters per second.

6. Column H cell H2 only, enter a value for rolling friction. Typical values are given for a track, a rough road, and a smooth road.

7. Column I: Enter the average power recorded during the measurement section.

8. Column J: Enter the time required to cover the measurement section for each trial.

9. When you have entered all these values the bike and rider CDA will appear in cell B12.

10. The goodness of fit of your data can be seen in cell Q9. That value should be around 0.98 or better.

Instructions for a Velodrome

The spreadsheet does not account for wind in a velodrome. Wind will tend to cancel for a complete lap, however these effects are nonlinear and you should do testing in the calmest conditions you can or on an indoor velodrome. The spreadsheet assumes a complete lap and your data might be improved by doing 2 or more laps. This will be difficult at high speed so you may want to use more laps for lower speeds and less laps at the higher speeds.

Start by going to the CnA on a velodrome tab. You will need to enter the following values one time only:

1. Cell B2: The length of the track in meters
2. Cell C2: The height of the saddle from the ground in meters.
3. Cell H2: Enter a value for rolling friction. Typical values are for a track are 0.0025.

You will need to enter the following values for each trial:

4. Column A: Bike and rider mass in kg.
5. Column D and E: Velocity and the beginning (initial) and end (final) of the measurement interval (e.g., one lap) in meters per second. These can be read off of the SRM data but you must know the time associated with the start and end.
6. Column F: Average velocity over the measured section in meters per second. This could be obtained from the SRM or calculated from the track length and the lap time.
7. Column I: Enter the average power recorded during the measurement section.
8. Column J: Enter the time required to cover the measurement section for each trial.
9. When you have entered all these values the bike and rider CnA will appear in cell B12.
10. The goodness of fit of your data can be seen in cell Q9. That value should be around 0.98 or better.

Reference


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